



Europäisches Patentamt
European Patent Office
Office européen des brevets



⑪ Publication number : 0 681 152 A2

⑫

EUROPEAN PATENT APPLICATION

⑬ Application number : 95830181.4

⑮ Int. Cl.⁶ : F25B 15/04

⑭ Date of filing : 05.05.95

⑯ Priority : 06.05.94 IT RM940283

⑰ Date of publication of application :
08.11.95 Bulletin 95/45

⑲ Designated Contracting States :
DE ES FR NL

⑳ Applicant : ENEA ENTE PER LE NUOVE
TECNOLOGIE, L'ENERGIA E L'AMBIENTE
Viale Regina Margherita, 125
I-00198 Roma (IT)

⑷ Inventor : Corallo, Giuseppe
Viale della Serenissima, 117
I-00177 Roma (IT)

⑸ Representative : Sarpi, Maurizio
Studio FERRARIO
Via Collina, 36
I-00187 Roma (IT)

④ Absorption heat pump systems for air conditioning.

⑤ A cold/warm absorption heat pump system for air conditioning including a vapour generator, an absorber, an evaporator, a pump of the enriched solution, and a pair of lamination valves connected to one another to form a closed circuit, is provided with an inherent safety device applied to a direct-flame generator such as to always keep the liquid level of the reboiler higher than a predetermined minimum level, a cold/warm switching valve controlling the system so that the latter operates in a reversible way even at reduced or half power without any switching off and restarting, a coolant level regulator adapted to keep the coolant between two predetermined levels for the optimum operation of the evaporator, and a "multi-pipes to pipe" absorber not subjected to the safety rules regarding the pressure reservoirs.

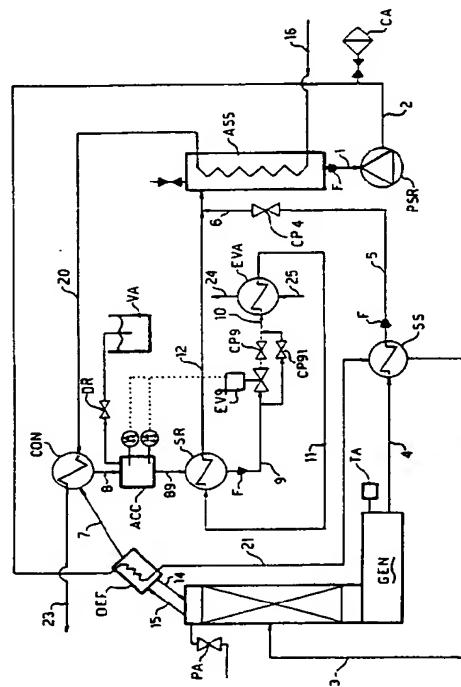


FIG. 6

The present invention relates to the field of the cold/warm air conditioning systems operating with a heat pump and, more particularly, concerns an improvement in the heat pump systems especially of the absorption type.

As known, the main feature of the heat pumps is the production of heat or cold in order to comply with the requests of the users.

Heat pumps divide into two wide classes, i.e. absorption and compression heat pumps, both using outer energy to cause the heat of the evaporator circuit at low temperature (-5/12°C) to pass to a higher temperature (45/60°C) so as to be transmitted to the users or to be dissipated. In case of absorption heat pumps the requested energy is thermal energy at high temperature (burner-vapour) and is used to heat the reboiler of a vapour generator which is at about 150-180°C.

From the foregoing it is apparent that heat pumps have two circuits with different functions, the first collecting heat at low temperature, the second transmitting the heat.

This invention aims at providing such a system in which several structural and functional teachings are introduced such as to achieve the dual object of avoiding mechanical complexities which cause the cost to increase and assuring in any case the requested performance, competitive final cost, and compliance with the technical rules in force in the main industrial countries.

This aim is achieved by providing

- a heat pump including a vapour generator, an absorber, an evaporator, a pump of the enriched solution, and a pair of lamination valves connected to one another so as to form a closed circuit;
- an inherent safety device to be applied to a direct-flame generator such as to always assure a level of the reboiler higher than a predetermined minimum value;
- a cold/warm switching valve controlling the system in a reversible way even at a reduced or half power without any switching off and restarting;
- a cooling fluid level regulator adapted to keep the level between two predetermined values so as to always assure the optimum operation of the evaporator; and
- a "multi-pipes to pipe" absorber which is not subjected to the safety rules for the pressure reservoirs.

The invention will now be described in a detailed way with reference to the accompanying drawings which show by way of an illustrative, non-limiting example a preferred embodiment of the invention. In the drawings:

Fig. 1 shows schematically an absorption heat pump system;

5 Figs. 2 and 2A show the detail of the safety device for the direct-flame vapour generator of Fig. 1; Fig. 3 shows the cold/warm diagram; Fig. 3A is the axial section of the switching valve having facing solid discs and used in the diagram of Fig. 3; Figs. 3B and 3C are two front views of the discs in the warm and cold positions, respectively; Fig. 4 shows two axial section views of the valve equipped or not with water-proof room; Figs. 5, 5A and 5B show three front views of an alternative switching disc also allowing the production of sanitary warm water in the warm, cold, sanitary warm water positions, respectively, according to the operating diagram of Fig. 10; Fig. 6 shows an alternative to the diagram of Fig. 1 in which a level regulator of condensed NH₃ is provided; Fig. 7 shows the detail of the level regulator in the storage tank of Fig. 6; Figs. 8 and 8a show elevation views of the absorber of Fig. 6 from the outside and in section, respectively; Fig. 9 shows the detail of the distributor associated to the absorber of Fig. 8; Fig. 10 is a block diagram showing the operating principle of the system according to the user's request; Figs. 11A and 11B show two different ways of connection of the components on the low-pressure side.

20 25 30 35 40 45 50 55

With reference to Fig. 1 the absorption heat pump consists of a series of main components connected to one another so as to form a closed circuit: vapour generator GEN, absorber ASS, evaporator EVA, enriched solution pump PSR, two lamination valves VR4 and VR9, dephlegmator DEF, condenser CON, subcooler SR, solution exchanger SS, and a burner B.

In such closed circuit, a particular fluid, for example a mixture of water and ammonia, is subjected to some cyclic transformations in order to transfer low-temperature heat (-5 to 12°C) from the evaporator to the condenser and the absorber at a higher temperature (about 45 to about 60°C) to the cost of high-temperature heat (160 to 180°C) supplied to the vapour generator.

Still referring to Figure 1, the solution enriched with ammonia from absorber ASS to line 1 is pumped by enriched solution pump PSR through line 2 to dephlegmator DEF to be cooled, and then is supplied to exchanger SS in order to regenerate heat and through pipe 3 to ammonia vapour generator GEN.

A solution 4 impoverished of ammonia flows from the bottom of the column generator GEN to exchanger SS where it is cooled and then through lamination valve VR4 to absorber ASS. A vapour current 15 enriched with ammonia flows from the top of generator

GEN to the upper dephlegmator and divides into a condensed liquid current 14, which is the reflux current for the enrichment section of the distillation column, and a vapour current 7 even more enriched of ammonia which condenses in condenser CON and then through line 8-89 is subcooled in subcooler SR, flows through line 9, is laminated in lamination valve VR9, reaches through line 10 evaporator EVA, where it largely evaporates again, flows through line 11 back to subcooler SR, where it absorbs sensitive heat, and then flows through line 12 to absorber ASS, where it restores the initial enriched solution, and begins a further cycle.

Low temperature heat is provided in the evaporator EVA at the cost of the glycol and water current of lines 25-24: "cold water" circuit.

The absorber is cooled by current 16 of glycol and water which also cools the condenser through line 20: "warm water" circuit. Generator GEN is heated by a high temperature source formed of fossil oil burner B. Pressures which establish within vapour generator GEN, i.e. about 18-25 ata, in case the coolant is ammonia and the absorbent liquid is water, fall in the domain of rules governing the construction and the operation.

Such rules concern all of the vapour generators having a volume greater than 5 litres and a little higher pressure than the atmosphere, as it occurs in most systems even of little power.

The rules in force provide such a lot of safety and operation standards that the commercial production of those systems having low and middle power is economically prohibitive.

The reason of such safety standards is that a burst may occur if the level of the liquid to be evaporated falls below the level licked by the flames. Such a burst is due to the fact that the part, which does not get wet, becomes red-hot and loses those mechanical properties which usually allow it to withstand the operating pressure.

In the absorption heat pumps, the pump of the enriched solution PSR provides for supplying the generator with the solution enriched with coolant (line 3) while the latter is emptied of the impoverished solution (line 4) and the distilled vapour (line 7).

In order that such a system gives economic advantages it is essential that the efficiency of the boiler is as high as possible, as it occurs in the generators with direct-flame heating which, however, are subjected to the most restrictive rules.

In order to avoid that the level of the liquid in the generator falls below the level reached by the flames due for example to a failure of the pump, the invention provides, in the line 4 of the impoverished solution as shown in Fig. 2, a passive safety device including a drum BT, which is run from the bottom to the top by the impoverished solution and communicates through pipe TE to the vapour generator column just

above the surface of the liquid under normal operating conditions, and a burst disc DR placed at a suitable height along the vapour generator column.

If the level of the liquid falls below the height of suction (Fig. 2A), the vapour from the top of drum BT prevents the impoverished solution from further flowing and emptying the reboiler and operates one of the arranged safety devices or the burst disc DR.

Under normal operating conditions the suction head is dipped in the liquid of the generator bottom and allows the impoverished solution to freely flow.

The size of the drum is very important for the effective safety operation of the device which operates in case of equipment failure causing the burst disc DR to open before the equipment reaches the bursting pressure.

With reference to Figs. 1, 2, 6, 7, under the worst condition when the generator would empty due to any reason while the burner is alight and the condenser is cool, a volume of liquid comprised between the two broken lines and designated by "u" is needed so as to provide for the further emptying which is only due to any evaporation through the pipe conveying the vapour to the condenser.

Such a volume of liquid should be at least the same as the volume of those components (condenser, one part of the storage tank and connecting pipes, i.e. the components of the closed circuit) which are normally emptied out of liquid and connected to the vapour output line before the coolant lamination valve, at the same pressure of the distillation system, which is in turn connected to those devices normally emptied out of liquid and free from density differences due to temperature and concentration.

Once the condenser is filled with liquid, a certain volume of liquid designated by "s" remains below the safety level and maintains the reboiler filled in that portion licked by the flames. At the same time the burst disc DR opens due to the sharp increase in the pressure.

The succession of the above-described events causes only a negligible quantity of condensate gradually filling the mentioned components to pass through the capillaries CP9 of coolant expansion, because a certain quantity of vapour passing from the suction pipe of the drum through the larger capillary of the impoverished solution would provide for compensating the pressure upstream and downstream of the capillary CP9.

If one does not rely upon the reached safety level, the drum may be arranged so that the volume designated by "u" is such as to fill not only the above-mentioned components but also those components downstream of the expansion capillaries which normally are not filled with liquid (evaporator, low-pressure side of the subcooler, absorber, connecting pipes), i.e. those components of the closed circuit which normally are emptied out of liquid and connected to the distillation

system.

It should be appreciated that, even if the part licked by the flames were partially emptied out of liquid, the high heat exchange coefficient of the boiling liquid would keep low the temperature of the metal walls of the reboiler so that the margin of safety of the described system is very high.

Advantageously, such an inherent safety device allows most of the departures from the above-mentioned rules in force to be achieved.

Fig. 3 shows the diagram of a system provided with an absorption heat pump PCA for air conditioning by means of convectors VC1 and VC2 by winter heat regeneration and summer heat disposal to the cost of the environmental air. Without explaining the logic of operation of the different temperature sensors and alarm devices, it should be noted that convectors VC1 operate on the inside and convectors VC2 on the outside in summer, vice versa in winter.

In order to understand the control logic of the auxiliary circuits of warm and cold water, reference should be made, beside Fig. 3, also to the block diagram of Fig. 10.

In summer, the system has to supply the cold to the rooms to be air conditioned and operates at the maximum power so that on/off electrovalve EV1 of the first burner B1 is open and the regulation is carried out by valve TC1 (set point at +7°C) modulating the gas flow rate with reverse proportionality.

In winter, the system operates at reduced power so that electrovalve EV1 is closed and the regulation is carried out by valve TC2 (set point at +50°C) modulating the gas flow rate with direct proportionality only to burner B2.

In order to automatically switch the system from summer to winter operation it is necessary to provide a device reversing the water currents in the positions designated by (*) in Fig. 3. Such a device should intercept eight pipes so that an eight-way switch is needed. In summer, convector VC1 operates to the inside, and VC2 to the outside, vice versa in winter.

The most simple embodiment of the reversal device performing the above-described function consists of two facing plane discs DF and DM (see Figs. 3A, 3B and 3C), one of which (DF) is fixed and receives fittings for pipes T, the other is movable (DM) and includes four slots C extending along circular sectors by 90°.

The reversal of the operation is carried out by a rotation of the movable disc by a quarter of revolution.

The winter position is shown in Fig. 3B, the summer position in Fig. 3C. The following symbols appear at the ends of the slots C and represent the eight pipes T to be connected:

IAC: input of warm water to the absorption heat pump PCA;

IAF: input of cold water to the absorption heat pump PCA;

IEX: input to the outside convector;

IIN: input to the inside convector;

UAC: output of the warm water from PCA;

UAF: output of the cold water from PCA;

UEX: output from the outside convector;

UIN: output from the inside convector.

As the pipe fittings are placed at 90° so as to be disposed in pairs at the ends of the engaged slots C, it is seen that the convector to the outside are connected to the cold water circuit and the convector to the inside are connected to the warm water circuit in winter, vice versa in summer.

In order to avoid a thermal short circuit among the currents supplying the multi-way valve, it is recommended to use materials having low thermal conductivity, such as synthetic resins, ceramic, glass, a.s.o.

Sealing O-rings are provided along the circular valve bodies and slots. Alternatively the sealing is assured by the contact between the plane surfaces of both discs because of the low pressure involved (5-10 mca).

The contact between the discs and the compensation of the wear clearance (see also Fig. 4) can be assured by an outer spring M coaxial with the control shaft AC integral with the movable disc and anchored to the fixed disc or by the water pressure due to the delivery head of one of the two circuits. In the latter case there is provided a sealed room CS integral with the fixed disc and including the movable disc. Such a room connects the surface of the movable disc opposite to that containing the slots C to one of the pipes at the highest pressure. Spring M is intended only to keep the rest position.

If a servomotor is used, the arranged eight-way valve may be used to automatically defrost the outer convectors under control of the negative pressure sensor DP1 by delivery of warm water caused by a short switching to the summer position and by stopping the circulator P1 of the evaporator. The warm water defrosts the outside convector VC2 and the cold water is prevented from being supplied to the inside convector VC1 to avoid an undesired lowering of temperature.

Flowmeter FA1 in the delivery circuit of the cooling side and the vacuum sensor DP2 along with the temperature alarm sensor TA at the bottom of the generator and the high-pressure alarm sensor PA operate to put out only the burner and to put on a warning light and an acoustic signal, if any, awaiting the manual reset of the safety block. If the waiting time is longer than a predetermined time the putting out of the whole system may be provided.

On the warm side of the circuit of Fig. 3, there are provided a storage tank AM to produce sanitary warm water and a three-way electrovalve EV2 to circulate the warm water only to the storage tank in case only the production of sanitary warm water is requested. As the passageways in the body of the multi-way

valve are easily provided, the function of the three-way valve in the above-described case may be extended by modifying the slots and by further rotating the disc from the winter position of Fig. 5. In Figs. 5A and 5B there are shown the positions of the valve in case of summer air conditioning and production of sanitary warm water, respectively.

The system shown in Fig. 6 is similar to that already described with reference to Fig. 1 and then the like references designate already described parts. In this figure there are provided a storage tank ACC, an expansion tank VA, an air chamber CA, capillaries CP, filters F, an electrovalve EV, a solution exchanger SS, and a burst disc DR.

In the present absorption systems operating for example with H_2O/NH_3 , the coolant is subjected to a subcooling before entering the evaporator in order to increase the efficiency of the system.

To this end, it is necessary to place between condenser CON and subcooler SR a second capillary or a lamination member to assure a further pressure difference which adds to the main pressure difference before the evaporator. Of course, such capillaries or lamination members are stationary components and do not allow the system to be perfectly regulated so as to operate at reduced power.

In order to avoid this drawback (see Figs. 6 and 7), there is provided downstream of the condenser a level regulator formed of a storage tank ACC before the subcooler SR and two capillaries CP9 and CP91 in parallel, one of which being intercepted by an electrovalve EV9.

The first capillary can accept a lower flow rate than the minimum flow rate produced by the system.

The second capillary together with the first one allows the passage of a greater amount of coolant than the maximum quantity produced by the system.

If electrovalve EV9 is controlled by two level switches LB44 and LB43, such a system operates as follows: I case: when the level in storage tank ACC reaches the maximum height, electrovalve EV9 should be opened and should be kept open until the minimum-level switch comes into operation.

II case: when the level in storage tank is at a minimum, the electrovalve is closed until the maximum level is reached again. In this way, the level is always comprised between the two predetermined minimum and maximum heights so as to assure the control for the condensing vapour from the condenser as well as the correct operation of the evaporator.

At the present state of art, the absorbers used in the absorption heat pumps are generally of the type providing a film falling on a pipe coil with glycol and water cooling. In order to simplify the construction and to avoid the already mentioned rules regarding the pressure reservoirs, the invention provides a "multi-pipes to pipe" absorber shown in Figs. 8 and 8a. The water-ammonia mixture flows within the multi-pipes MT.

The cooling occurs outside such multi-pipes in the room SZ but inside the tubular jacket TU of the pipe coil. Heat exchanger TU is connected downstream of a nebulizer D (Fig. 9) mixing the phases before their entering the multi-pipes. Such a nebulizer has a cylindrical body CL in which the liquid-vapour spray mixture is conveyed to the direction indicated by the arrow.

After the distributor-nebulizer D there is a fitting RM which divides into five equal parts the current after the mixing and conveys them to the already described pipes MT through the fitting RC.

A tank RA may be provided at the output of the enriched solution (Fig. 8) in case the upper level of absorber ASS is higher than the lower level of the subcooler SR so as to collect the liquid 12L from the liquid-vapour separator SLV downstream of the subcooler SR (Fig. 11B).

The present invention has been illustrated and described according to a preferred embodiment thereof, however it should be understood that construction modifications may be made by those skilled in the art without departing from the scope of the present industrial invention.

25

Claims

1. An absorption heat pump system for air conditioning, in which a direct-flame vapour generator, an absorber, an evaporator, a pump of the enriched solution, a pair of lamination valves, a subcooler, and a solution exchanger are connected to one another so as to form a closed circuit, wherein it further includes in combination:
 - inherent safety means for the components exposed to the direct flames of the burner;
 - a cold/warm switching valve adapted to operate under reversible control the system even at reduced or half power without any switching off and re-starting;
 - a cooling fluid level regulating means adapted to keep such level between two predetermined values so as to always assure the optimum operation of the evaporator; and
 - a multi-pipe absorber, said pipe being contained in a tubular coil jacket.
2. The system according to claim 1, characterized in that said inherent safety means is formed of a drum filled with a suitable quantity of liquid solution and connected to the reboiler so as to prevent the latter from being accidentally emptied out below a predetermined safety level.
3. The system according to claims 1 and 2, characterized in that said drum is placed at the bottom

output of the reboiler, is provided with a pipe for its connection above the surface of the liquid of the reboiler to compensate the levels, and is also provided with a suction head so as to stop the current of the liquid when the remaining volume of liquid is equal to that of all the normally empty components of the closed circuit connected thereto at the same pressure of the distillation system.

4. The system according to claim 3, characterized in that the volume of the liquid is at least equal to that of the other normally empty components of the closed circuit connected to the distillation system.

5. The system according to claims 1 to 4, characterized in that said switching valve is formed of a pair of facing discs, the first of which is fixed and the second is movable, said first disc having eight holes and the second disc having two pairs of slots shaped as circular sectors forming as a whole an eight-way valve.

10

6. The system according to claims 1 to 5, characterized in that said eight-way valve switches the warm and the cold circuits according to the positions of the movable disc with respect to the fixed disc as well as according to a third position in which only sanitary warm water is produced in case a different geometry of the slots is provided.

15

7. The system according to claims 1 to 6, characterized in that the sealing between the fixed disc and the movable disc is provided by elastic gaskets or by the delivery pressure of one of the two circuits causing the facing discs to press each other.

20

8. The system according to claims 1 to 7, characterized in that said coolant level regulating means is formed of a storage tank upstream of the subcooler, and a pair of capillaries in parallel, one of which is intercepted by an electrovalve.

25

9. The system according to claims 1 to 8, characterized in that the first capillary can accept a flow rate lower than the minimum generated flow rate, and the second capillary together with the first capillary can accept a flow rate of the coolant higher than the maximum flow rate generated by the system.

30

10. The system according to claims 1 to 9, characterized in that said electrovalve is operantly associated to two level switches so as to keep within said storage tank a level of coolant always be-

35

40

45

50

55

tween two predetermined limit levels.

11. The system according to claims 1 to 10, characterized in that the absorber is formed of a "multi-pipes to pipe" exchanger including a bundle of pipes non-contacting to one another and placed within a tubular jacket shaped as a pipe coil or spiral having a vertical axis.

12. The system according to claims 1 to 11, characterized in that the two-phase mixture forming the absorption phase (water-ammonia) flows in parallel in said bundle of pipes with downward helicoidal movement, while the coolant flows in counter-current in the room among the pipes and the outer tubular jacket.

13. The system according to claims 1 to 12, characterized in that the absorber is fed by a distributor having a hollow cylindrical body, in which the liquid-vapour spray mixture is conveyed, and by a fitting-distributor placed at the end of said cylindrical body and provided with several radial passageways connected to the pipes of the bundle.

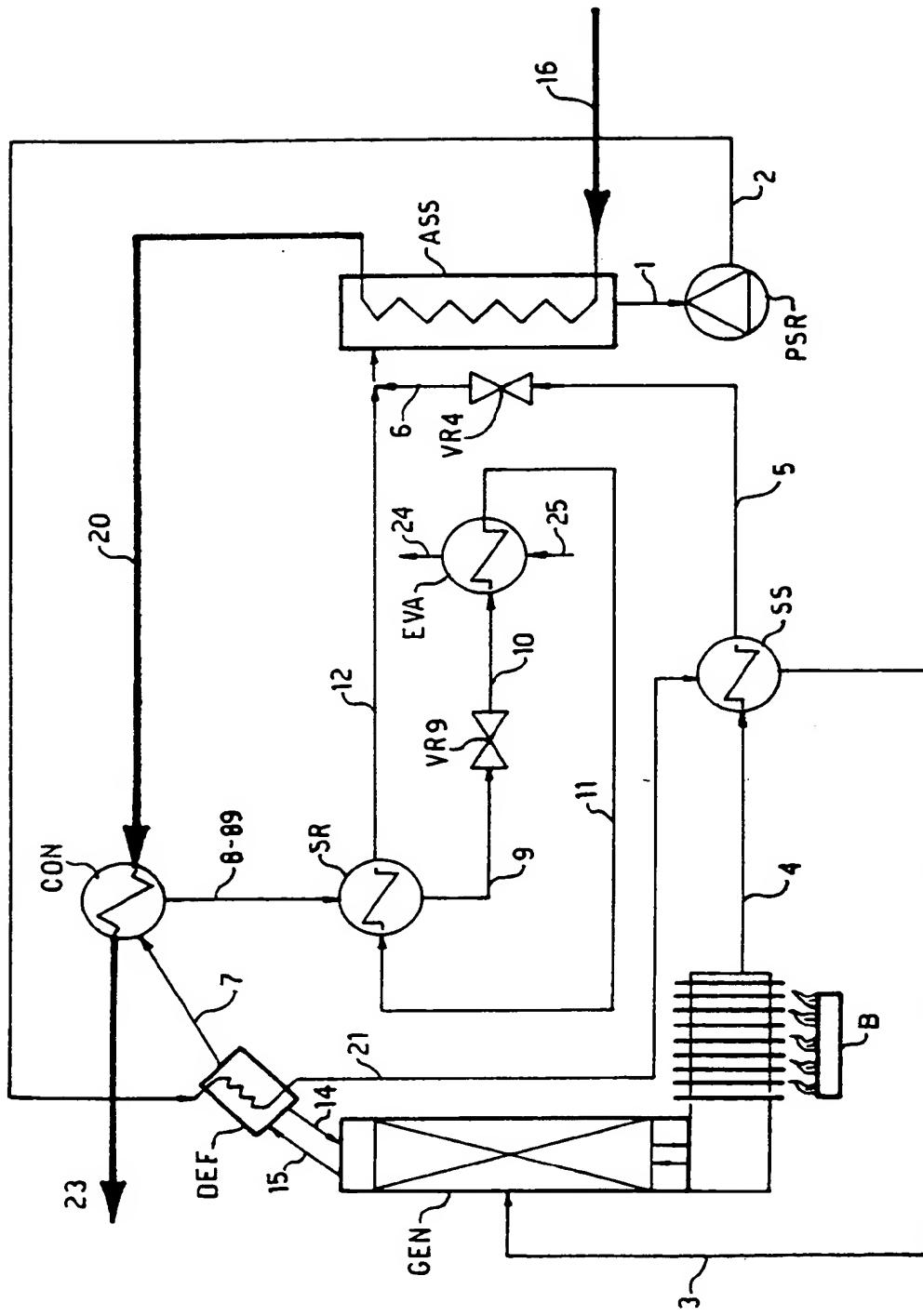


FIG. 1

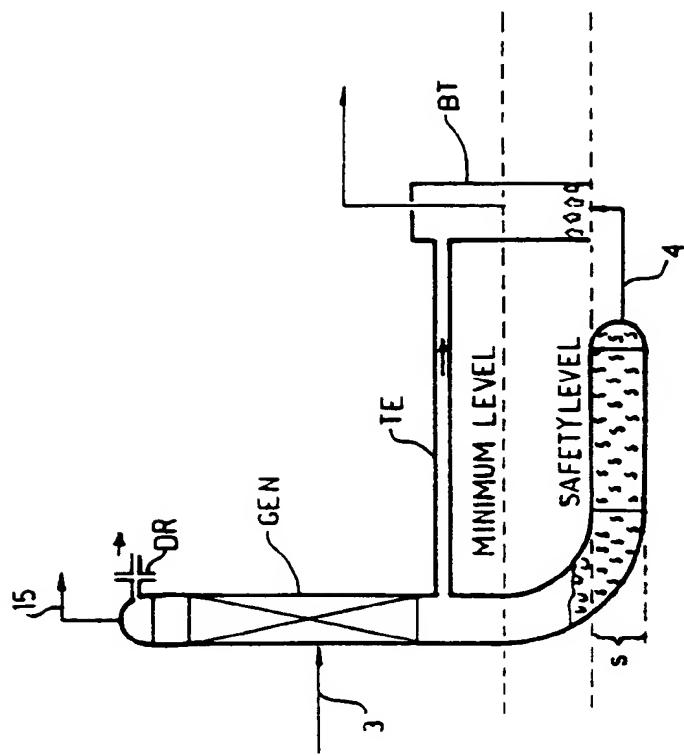


FIG. 2A

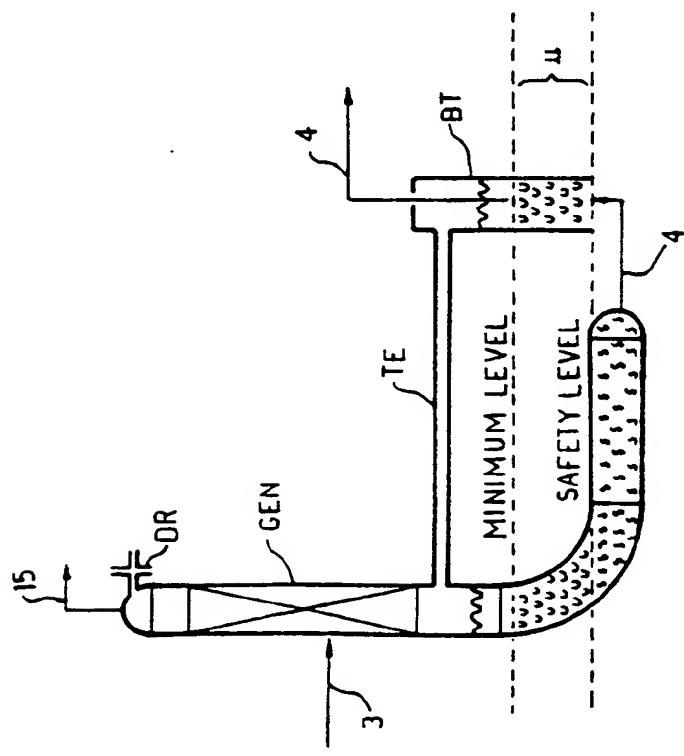


FIG. 2

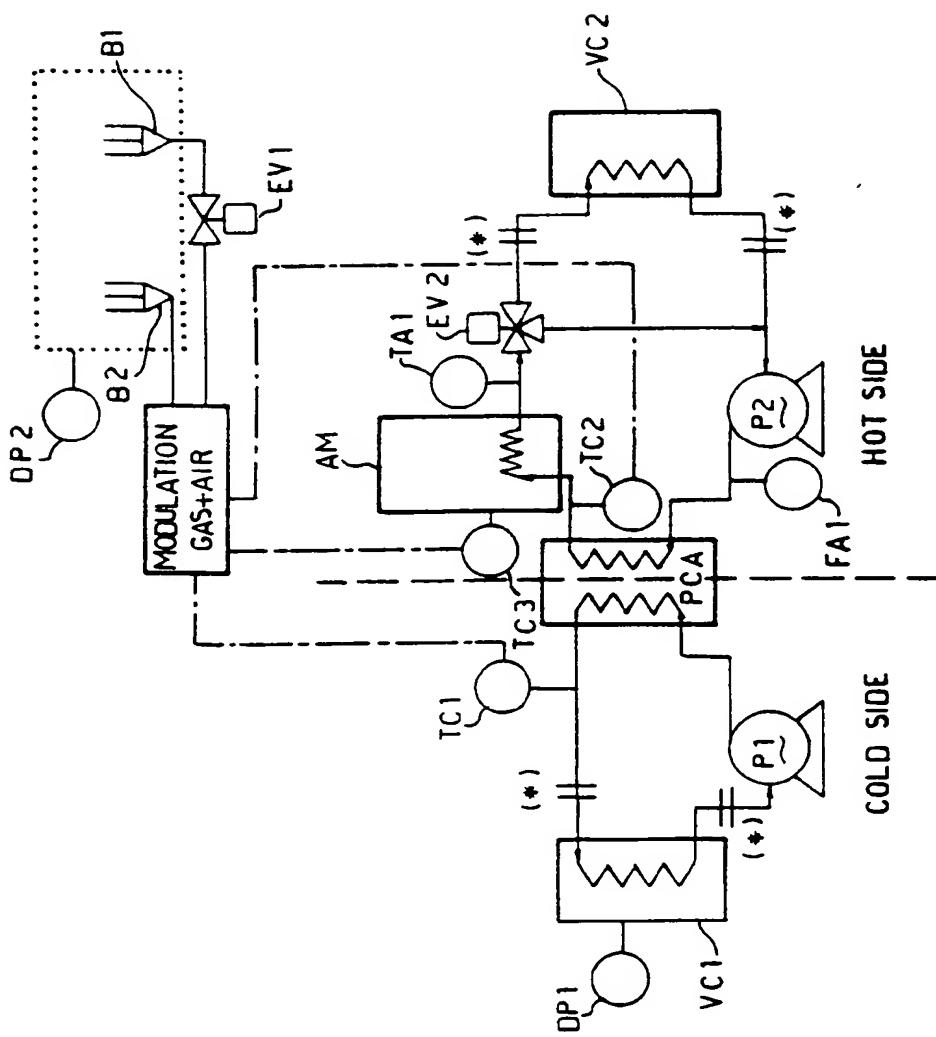


FIG. 3

FIG. 3C

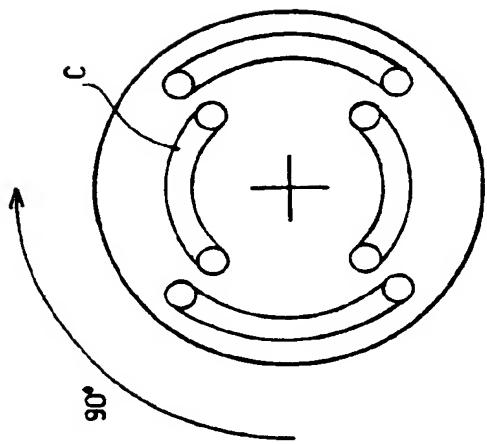


FIG. 3B

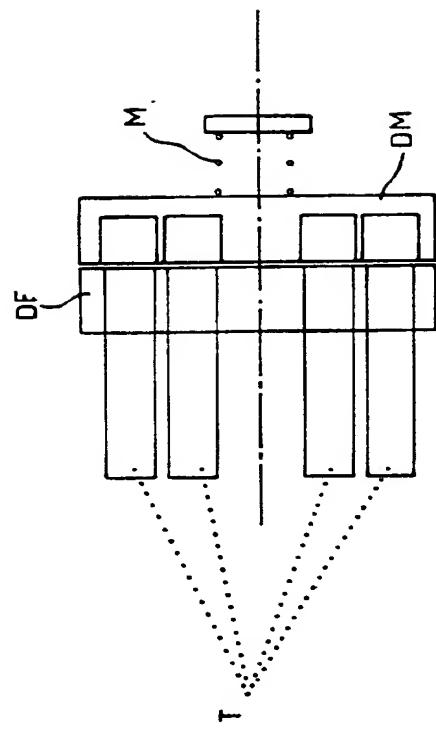
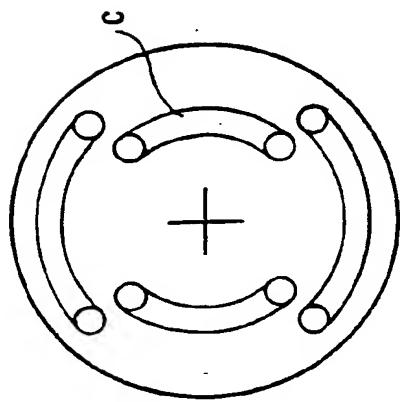


FIG. 3A

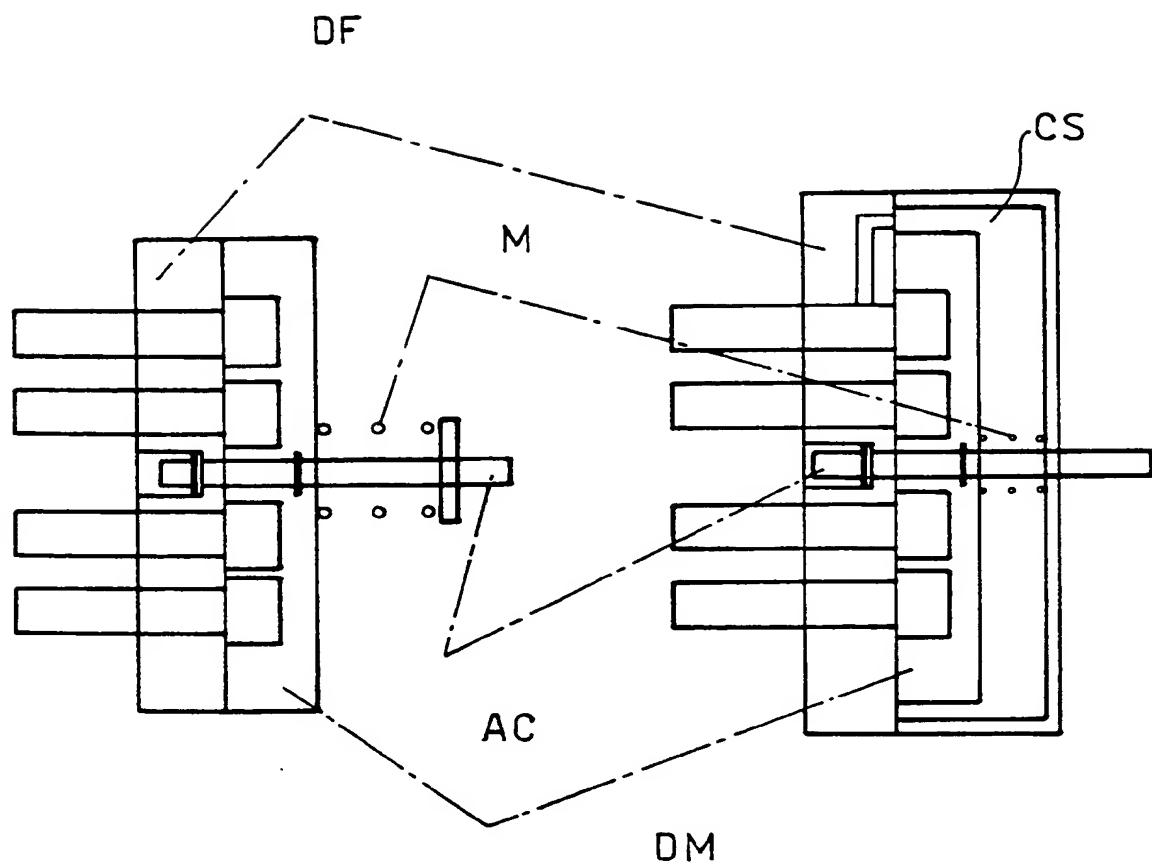


FIG. 4

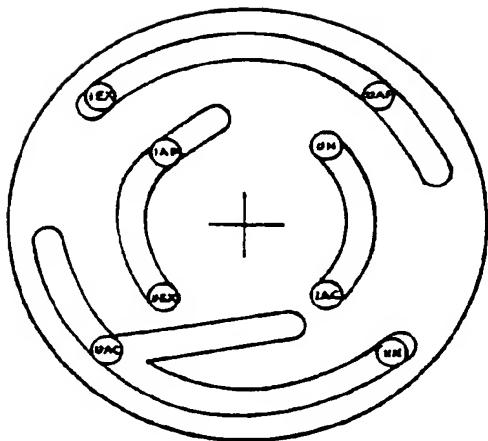


FIG. 5

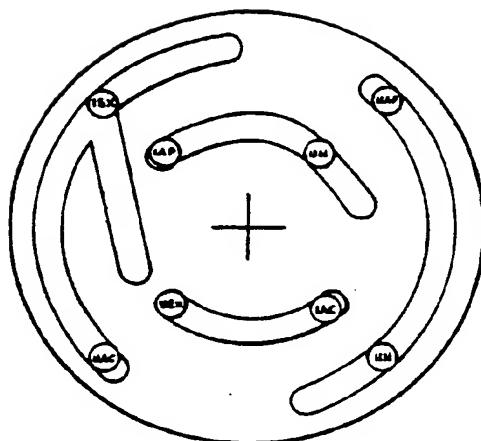


FIG. 5A

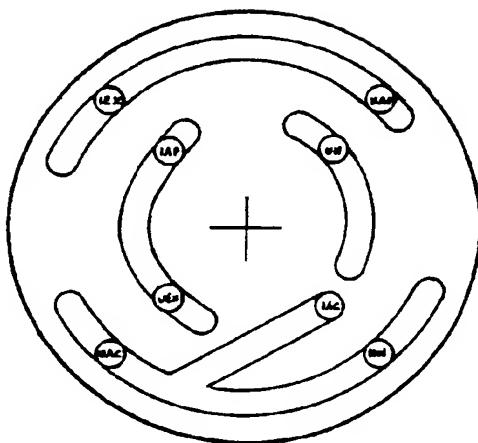


FIG. 5B

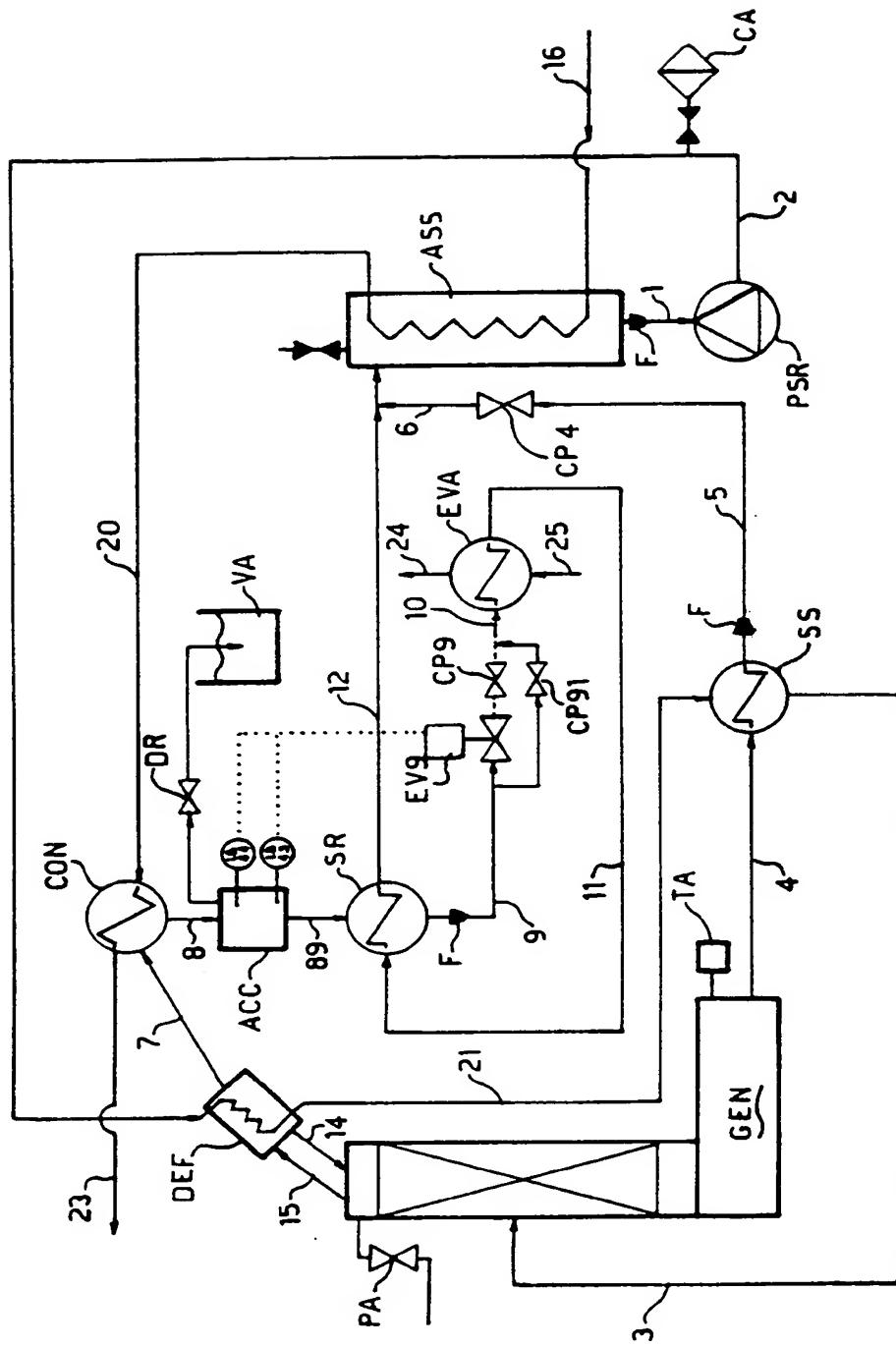


FIG. 6

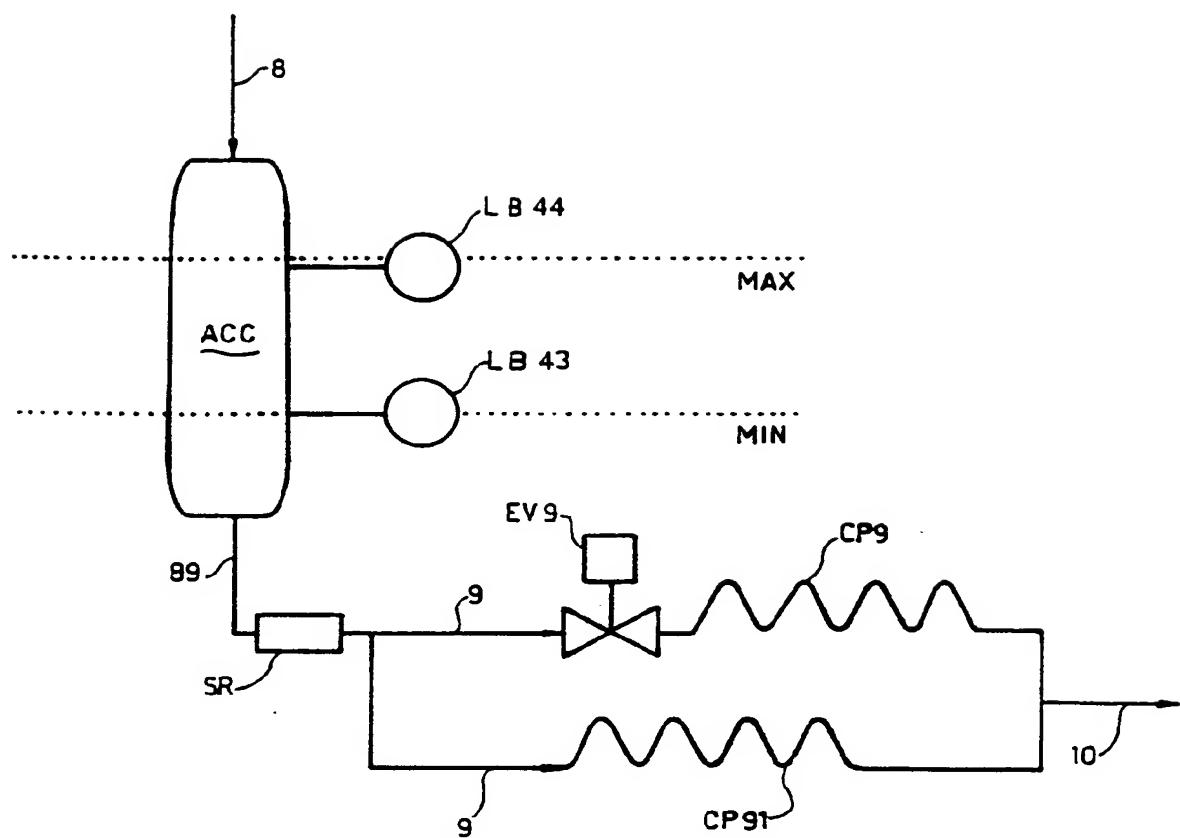


FIG. 7

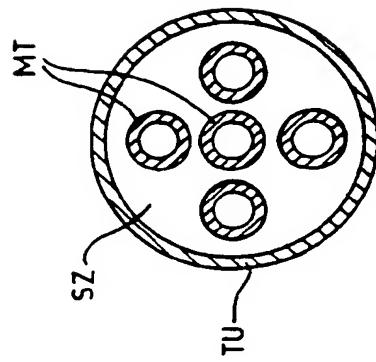


FIG. 8A

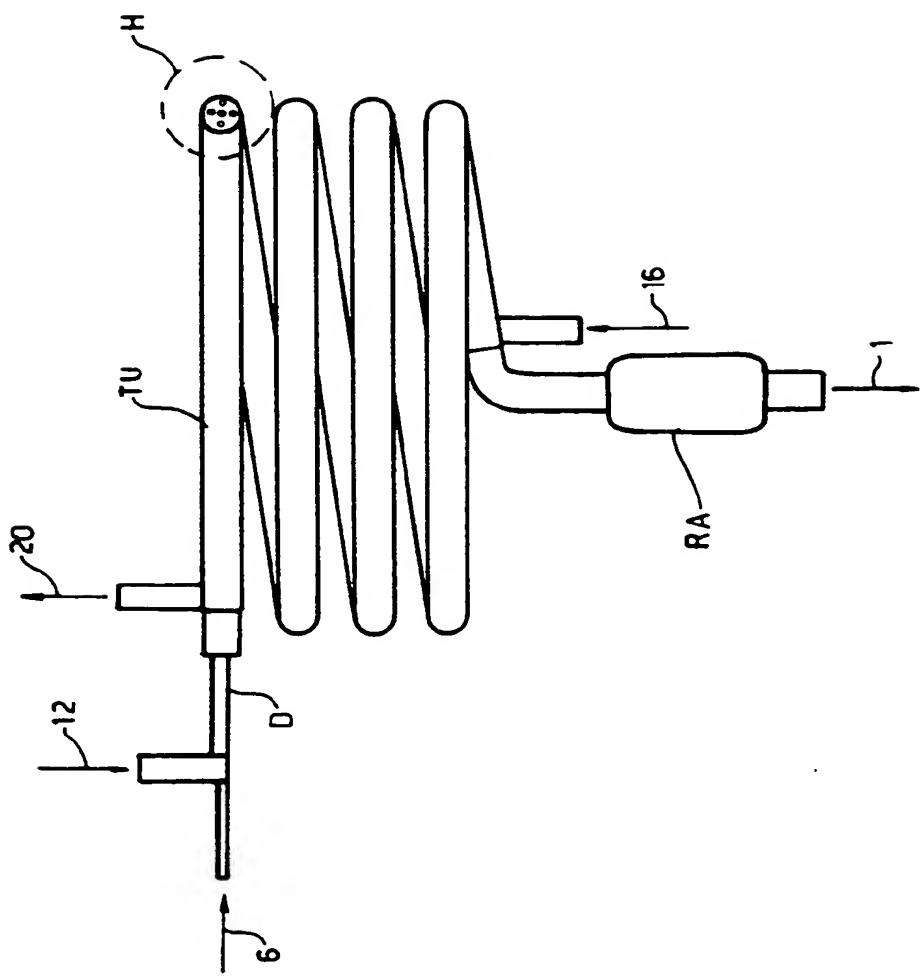
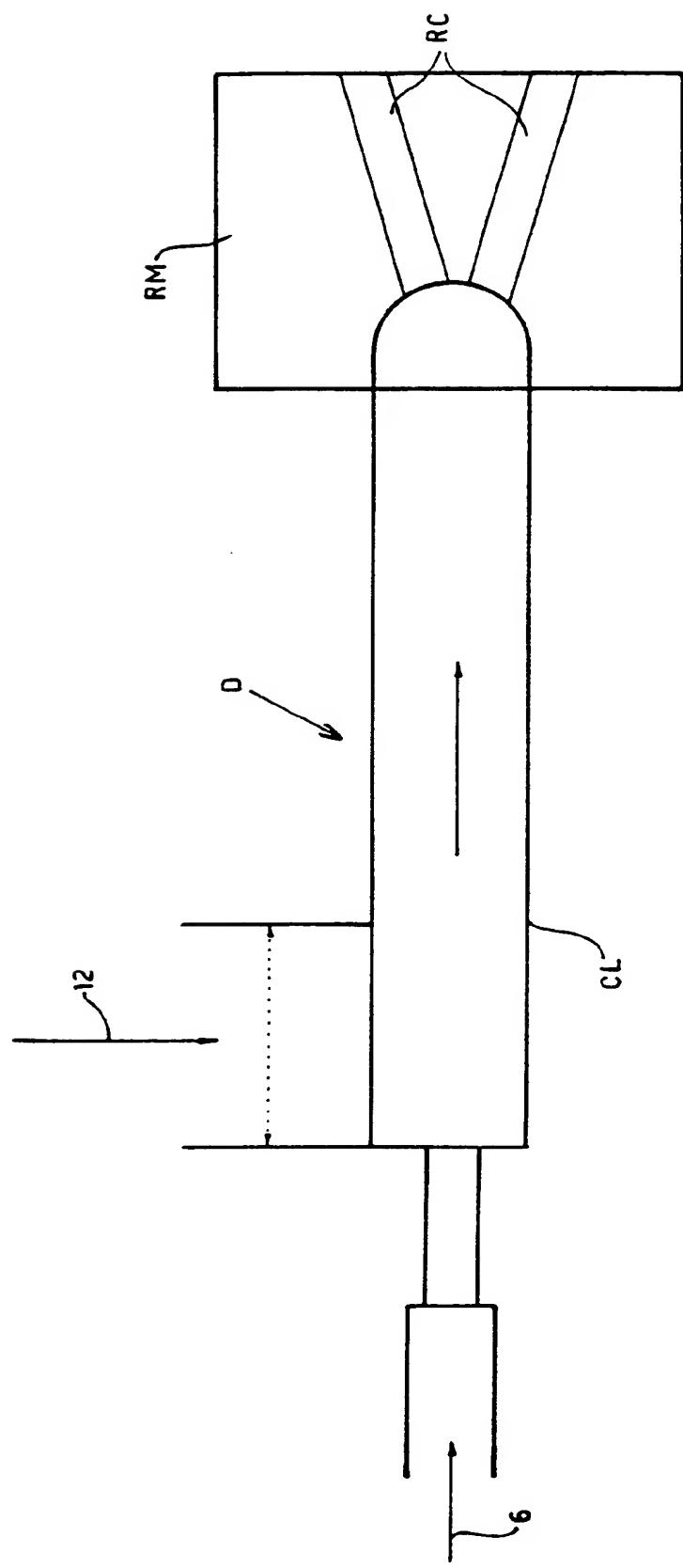


FIG. 8



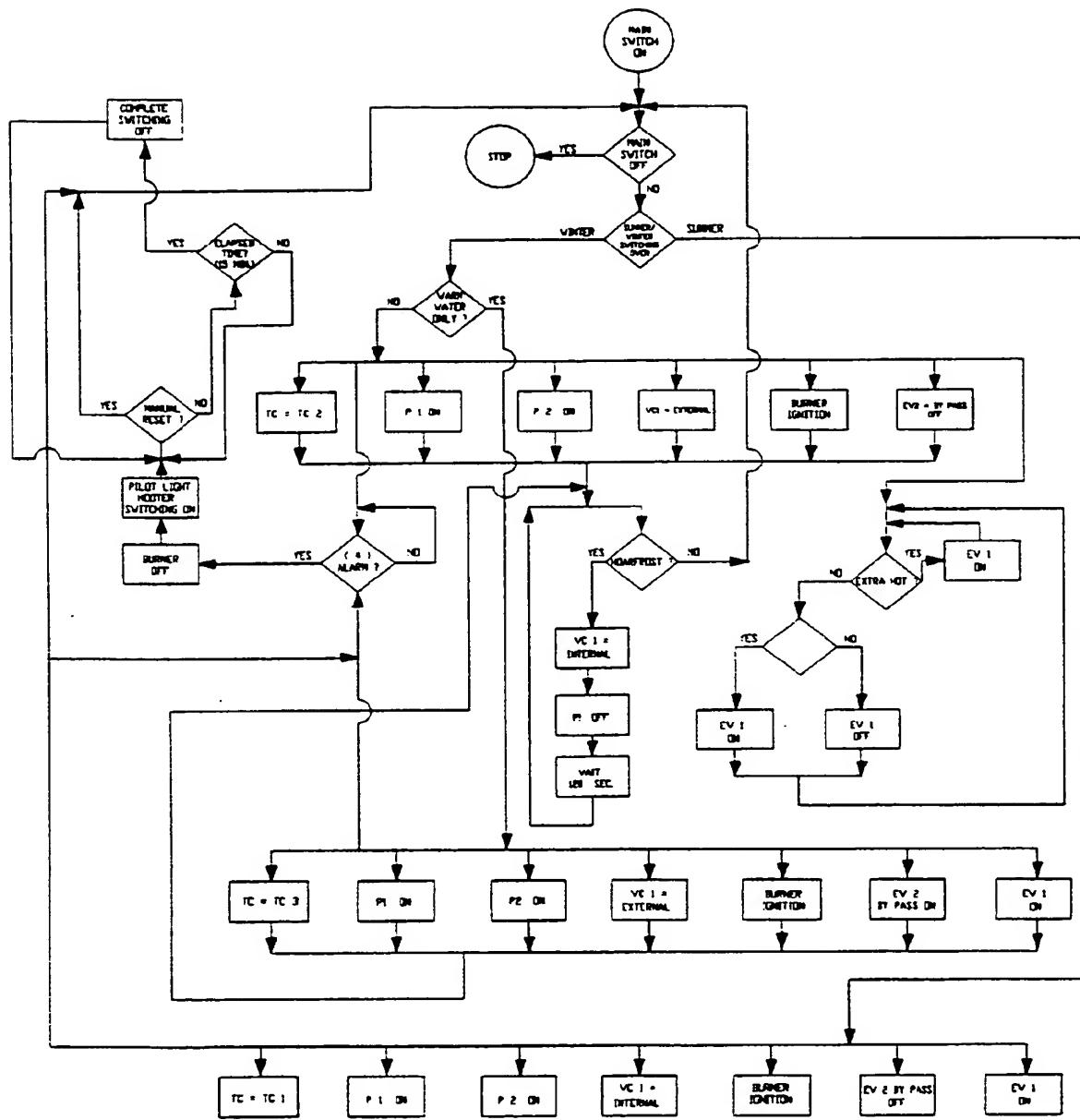


FIG. 10

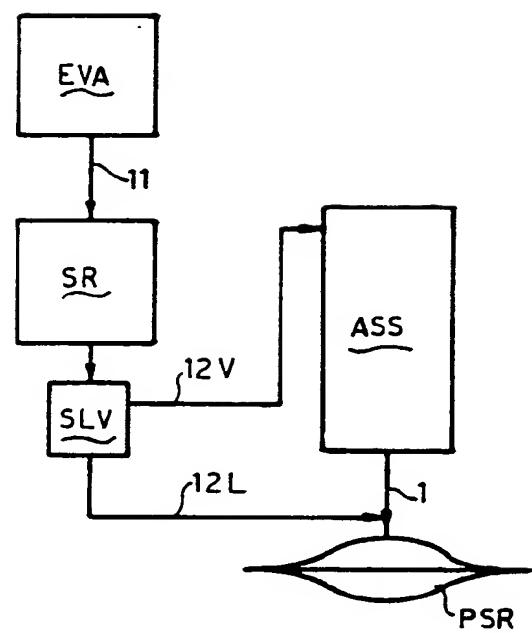
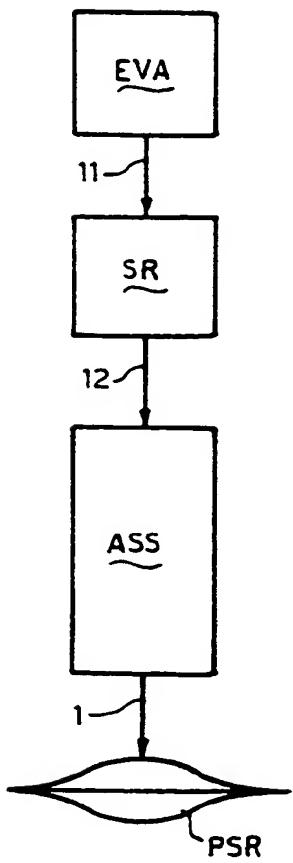
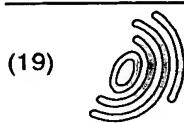


FIG. 11A

FIG. 11B



(12)

EUROPEAN PATENT APPLICATION

(88) Date of publication A3:
11.06.1997 Bulletin 1997/24

(51) Int Cl. 6: F25B 49/04, F25B 15/04

(43) Date of publication A2:
08.11.1995 Bulletin 1995/45

(21) Application number: 95830181.4

(22) Date of filing: 05.05.1995

(84) Designated Contracting States:
DE ES FR NL

(72) Inventor: Corallo, Giuseppe
I-00177 Roma (IT)

(30) Priority: 06.05.1994 IT RM940283

(74) Representative: Sarpi, Maurizio
Studio FERRARIO
Via Collina, 36
00187 Roma (IT)

(71) Applicant: ENEA ENTE PER LE NUOVE
TECNOLOGIE, L'ENERGIA E L'AMBIENTE
I-00198 Roma (IT)

(54) Absorption heat pump systems for air conditioning

(57) A cold/warm absorption heat pump system for air conditioning including a vapour generator (GEN), an absorber (ASS), an evaporator (EVA), a pump (PSR) of the enriched solution, and a pair of lamination valves (VR4,VR5) connected to one another to form a closed circuit, is provided with an inherent safety device (DR, BT) applied to a direct-flame generator (GEN) such as to always keep the liquid level of the reboiler higher than

a predetermined minimum level, a cold/warm switching valve (DF,DM) controlling the system so that the latter operates in a reversible way even at reduced or half power without any switching off and re-starting, a coolant level regulator (ACC,CPS,CP1) adapted to keep the coolant between two predetermined levels for the optimum operation of the evaporator (EVA), and a "multi-pipes to pipe" absorber (TV,MT) not subjected to the safety rules regarding the pressure reservoirs.

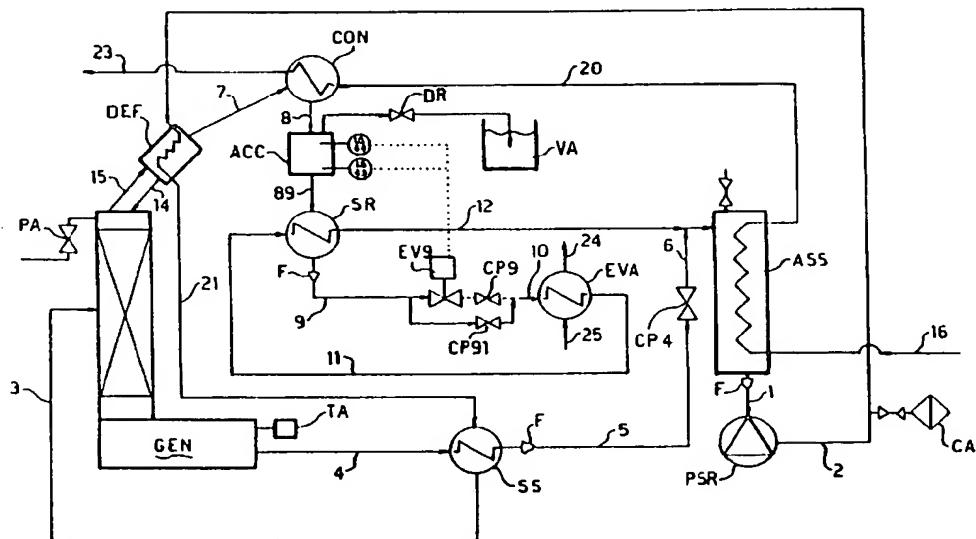


FIG. 6



European Patent
Office

EUROPEAN SEARCH REPORT

Application Number

DOCUMENTS CONSIDERED TO BE RELEVANT			EP 95830181.4						
Category	Citation of document with indication, where appropriate, of relevant passages	Relevant to claim	CLASSIFICATION OF THE APPLICATION (Int. Cl. 6)						
A	<u>US - A - 5 289 868</u> (KOSEKI) * Claims * --	1	F 25 B 49/04 F 25 B 15/04						
A	<u>EP - A - 0 066 277</u> (ASK WÄRMEPUMPEN) * Totality * --	1							
A	<u>US - A - 4 719 767</u> (REID) * Totality * --	1							
A	<u>DE - A - 3 149 781</u> (RUHRGAS) * Totality * --	11,12							
A	<u>US - A - 4 578 961</u> (DOMNICK) * Totality * ----	12							
			TECHNICAL FIELDS SEARCHED (Int. Cl. 6)						
			F 25 B						
<p>The present search report has been drawn up for all claims</p> <table border="1" style="width: 100%; border-collapse: collapse;"> <tr> <td style="width: 33%;">Place of search</td> <td style="width: 33%;">Date of completion of the search</td> <td style="width: 34%;">Examiner</td> </tr> <tr> <td>VIENNA</td> <td>02-04-1997</td> <td>WITTMANN</td> </tr> </table>				Place of search	Date of completion of the search	Examiner	VIENNA	02-04-1997	WITTMANN
Place of search	Date of completion of the search	Examiner							
VIENNA	02-04-1997	WITTMANN							
<p>CATEGORY OF CITED DOCUMENTS</p> <p>X : particularly relevant if taken alone Y : particularly relevant if combined with another document of the same category A : technological background O : non-written disclosure P : intermediate document</p> <p>T : theory or principle underlying the invention E : earlier patent document, but published on, or after the filing date D : document cited in the application L : document cited for other reasons & : member of the same patent family, corresponding document</p>									